

# CIERVA MEMORIAL PRIZE ESSAY COMPETITION, 1955

## THE PRIZE ESSAY

### **“On the Generalized Simple System for Automatic Stabilisation of a Helicopter in Hovering Flight”**

by

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#### SUMMARY

This paper deals with the instability of a hovering helicopter with controls fixed. The generalized simple stabilization system, which is composed of rods, springs, dampers and masses and uses the rotor shaft to generate gyroscopic forces, has been analysed. Special cases of a Second Order System have been considered and show that good stability can be achieved, thus overcoming the limitations of First Order Systems.

#### 1 Introduction

One of the helicopter's greatest assets is its ability to hover. However, in this condition it is dynamically unstable with the controls fixed. In other words, a disturbance from the equilibrium position grows with time. This means that the aircraft controls must be operated continuously with resulting pilot fatigue and increased possibility of accident.

The alleviation of this instability has been investigated over the past few years. Searches have been made for an automatic control device which could be fitted to the helicopter such that, with the controls fixed, a disturbance would decay and the aircraft return to its equilibrium position. Of such devices there are those that use as the basis of gyroscopic couples the rotor shaft in contrast to other methods where a separate gyroscope has to be installed. At the present time there exist two types which belong to the first category, they are the Bell Stabilizing Bar and the Hiller Servo Blade control. Although they do improve the stability they are not completely satisfactory. At best they give neutral stability. In this paper the principle of their mechanism has been generalized and the possibilities of a more satisfactory device have been investigated.

Attention has been focussed mainly on the Sikorsky R-4B since for this configuration the results can be compared with past investigations. We have considered too only the longitudinal motion for it is known that if the criterion for longitudinal stability is satisfied then that for the lateral motion follows.

The results are given in terms of the automatic control component in phase with the attitude, that in phase with the rate of change of attitude and their ratio. For convenience the dimensionless term  $\theta_q \Omega$  is used. From these results the stability characteristics are deduced.

## 2 *Stability in Hovering Flight*

Sissingh in reference 1 has shown that by considering a helicopter of arbitrary design and allowing it to have two degrees of freedom, *viz*, attitude and horizontal linear velocity, the equation of longitudinal motion lead to a frequency equation of the form

$$A_3 v^3 + A_2 v^2 + A_1 v + A_0 = 0 \quad (2.1)$$

where the A's are known for a particular aircraft configuration. The disturbances in the degrees of freedom are assumed to be of exponential form and the coupling between lateral and longitudinal motion has been neglected. Given also are stability charts so that the times to either double or halve the amplitude of the oscillations together with the period can be determined for given values of the constants  $A_1$  in equation (2.1).

Attention is focussed mainly on the Sikorsky R-4B and for this type the value of  $A_1/A_3$  is nearly zero. From the stability charts it can be seen that the helicopter is unstable and that if  $A_1/A_3$  could be increased stability would be improved.

If we introduce a hypothetical autopilot which will impose upon the mean blade setting  $\theta_0$  a cyclic pitch  $-(\theta_a \alpha + \theta_q q) \sin \psi$  the constants of the frequency equation  $A_1$  and  $A_2$  now become functions of the arbitrary parameters  $\theta_a$  and  $\theta_q$  respectively. Physically this procedure corresponds to a cyclic pitch variation which is dependent on the attitude and the rate of change of attitude with time.

Reference 1 shows that suitable stability can be obtained for  $\theta_a = 0.12$  and  $\theta_q \Omega = 3.25$ . This converts the unstable helicopter into one where the disturbance is halved in 3 secs with period 11 secs.

It is the purpose of the generalized simple stabilization system to use the rotor shaft as the basis of gyroscopic forces in order to obtain the required values of  $\theta_a$  and  $\theta_q \Omega$ .

## 3 *The Generalized 'Simple' Stabilization System*

Unlike fixed wing aircraft the helicopter does not have to be installed with a special gyroscope in order to obtain gyroscopic couples, for it can utilize its own rotor. Let us consider therefore a system which is fitted to and rotating with the rotor shaft and which can transmit the required cyclic pitch variation to the rotor blades to give good stability following a disturbance from equilibrium. In dealing with 'simple' systems we are considering ones where masses, springs, rods and dampers are used as the fundamental units of the control mechanism. The simple system is attached to the shaft and a rod leads from the control device to a mechanism at the hub.

which varies the pitch of the blades. Following a disturbance of the rotor shaft therefore a change in angle  $\Delta\theta$  will be effected. The system is represented diagrammatically in Fig 1.

When the shaft is in its initial position ( $t = 0$ ) an arbitrary point on this connecting rod will be at  $\xi_0$  (say). If following a disturbance the mechanism were allowed to roll over the rotor without hindrance the arbitrary point would have moved to  $\xi_1$ . Owing to the gyroscopic forces, the control device does not follow the rotor immediately. Let us assume the position of the point be at  $\xi_t$  after time  $t$  secs.

The response therefore to the disturbance is

$$\delta = (\xi_t - \xi_1) \quad (3.1)$$

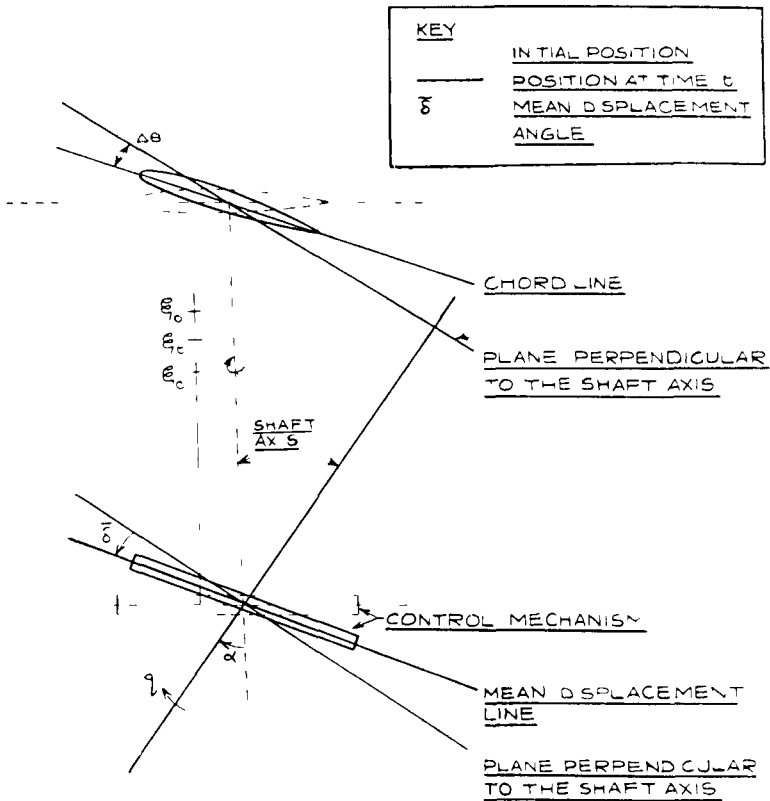
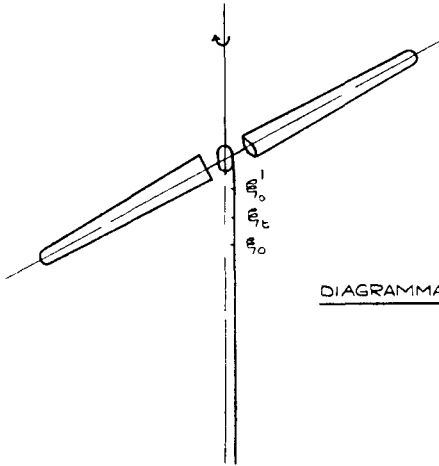


FIG 1 DIAGRAMMATIC REPRESENTATION OF THE  
GENERALISED SIMPLE SYSTEM

We shall suppose that the mechanism governing the actual pitch variation acts in a manner such that

$$\Delta\theta = G\delta \quad (3.2)$$

where  $G$  is a constant and that for a two-bladed rotor the pitch variations are equal but of opposite sign (see Fig. 2)



DIAGRAMMATIC REPRESENTATION OF THE  
HUB MECHANISM

FIG. 2

We shall consider the generalized 'simple' system to be composed of  $n$  rods hinged at the axis of rotation and connected to some form of mass. The mass may or may not be designed to have any special aerodynamic characteristics. The rods are damped at the hinge, between which, and the rotor shaft, springs are attached. The rods are also mutually interlinked by springs and dampers and are connected to the main control rod which feeds the response into the mechanism governing the blade pitch angle.

The automatic control response is defined by

$$\delta = \sum_{i=1}^n n_i \delta_i \quad (3.3)$$

where  $n_i$  are constants and the  $\delta_i$ 's are the angular displacements of the respective rods.

By considering the equation of motion about the hinge of the  $i$ th rod taking into account aerodynamic, mass, hinge damping and interconnecting moments and neglecting feedback due to the other rods through and due to the attachment to the main control, as well as the inertia of the blades about their longitudinal axis we obtain  $n$  equations of the form

$$\begin{aligned} \delta_i + 2\Omega \sum_{j=1}^n A_{ij} \delta_j + \Omega^2 \sum_{j=1}^n B_{ij} \delta_j + 2\Omega q \sin(\psi + \psi_i) \\ - q \cos(\psi + \psi_i) - 2qK_i \Omega \cos(\psi + \psi_i) = 0 \end{aligned} \quad (3.4)$$

where  $i = 1, 2, \dots, n$

The derivation of these equations is given in Appendix I. It is a simple extension to prove that the form of the equations is unaltered if we consider additional rods which instead of being attached to the shaft are hinged at points along the present rods.

It is assumed too that the moments due to the linking of the rods by springs and dampers act in the same plane as the other moments. If therefore two rods, 1 and j, are at a large angle the interlinking moment would act in a plane nearly orthogonal to the planes of the other moments. This

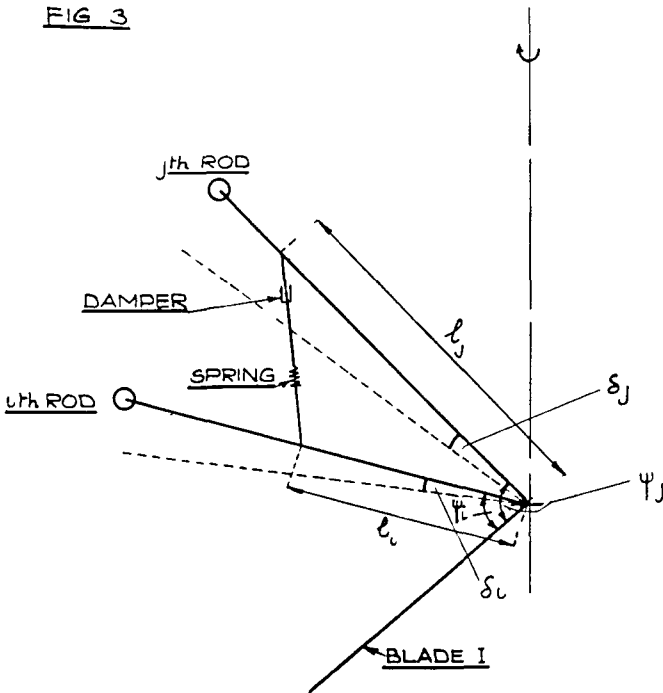


ILLUSTRATION OF THE CONNECTION BETWEEN THE  
lth AND jth RODS

has no advantage from the response point of view and it may provide structural worries. If therefore  $|\psi_l - \psi_j|$  is large the appropriate coefficients in equation (3.4) will be small (see Fig. 3).

Equations (3.2-4) define the response of the control system to a disturbance in pitch of the helicopter in the hovering condition. In general  $\Delta\theta$  will be of the form

$$\Delta\theta = \theta_s \sin \psi + \theta_c \cos \psi \quad (3.5)$$

with  $\theta_s, \theta_c$  non-zero. This corresponds to displacements in the longitudinal and lateral control. The lateral component  $\theta_c$  will be neglected,

$ze$ , we shall neglect coupling between the two motions. This lateral component can in fact be used to compensate for the already existing coupling.

The component in the longitudinal plane  $\theta$  is itself composed of two parts. Firstly a component in phase with the change of attitude and secondly a part in phase with the rate of change of attitude with time.

Adopting the frequency response technique we consider the response to a sinusoidal variation in  $\alpha$

$$\alpha = \alpha_0 \sin vt \tag{3.6}$$

We can then obtain on solving equations 3.2—6

$$\theta_\alpha = \theta_\alpha(v, A_{11}, B_{11}, K_1, \psi_1, n_1, G, \Omega) \tag{3.7}$$

and

$$\theta_q = \theta_q(v, A_{11}, B_{11}, K_1, \psi_1, n_1, G, \Omega) \tag{3.8}$$

The method is itself a first approximation. We should strictly consider a decreasing or increasing oscillation for  $\alpha$  but the above assumption (3.6) simplifies the mathematics involved and investigations have shown (Ref. 1) that the introduction of a varying amplitude has only a small effect on  $\theta_\alpha$  and  $\theta_q$ .

Equations 3.7—8 show that  $\theta_\alpha$ ,  $\theta_q$  are functions of the parameters of the system. By their appropriate choice we can obtain the desired values of  $\theta_\alpha$ ,  $\theta_q$  to give the type of stability required. In this paper we are looking at the problem from the theoretical viewpoint and it will be shown later that the desired values of  $\theta_\alpha$ ,  $\theta_q$  can be achieved. In practice, however, the constants of the system will not be independent but will functionally be related by either engineering limitations or factors intrinsic to a proposed design. Here we are content to show that good stability is possible using simple systems as defined.

No attempt has been made to solve the equation 3.2—6 generally but concentration has been fixed on particular values of  $n$ ,  $ze$ , for systems with given degrees of freedom.

Let us first consider the simplest case

#### 4 First Order Systems ( $n = 1$ )

For a system with one degree of freedom the equation of motion becomes

$$\delta + 2A_{11}\Omega\delta + \Omega^2B_{11}\delta + 2\Omega q \sin(\psi + \psi_1) - q \cos(\psi + \psi_1) - 2K_1q\Omega \cos(\psi + \psi_1) = 0 \tag{4.1}$$

and the change in the pitch angle of the blade is given by

$$\Delta\theta = G\delta = -(\theta_\alpha\alpha + \theta_qq) \sin\psi \tag{4.2}$$

The systems in current use can be divided into two classes. Firstly there are those that incorporate only mechanical damping at the hinge and little aerodynamic damping and secondly there are those where the aerodynamic moment forms the major contribution.

The first case is the well-known Bell Stabilizing Bar and in effect is governed by the equation

$$\delta + 2K\delta\Omega + \Omega^2\delta + 2\Omega q \cos \psi + q \sin \psi = 0 \quad (4.3)$$

*i.e.*  $K_1 = 0, B_{11} = 1, A_{11} = K, \psi_1 = \pi/2$

The second class is typified by the Hiller Servo Blade Control and the equation of motion here becomes

$$\delta + 2K\delta\Omega + \Omega^2\delta + 2\Omega q \cos \psi + q \sin \psi + 2Kq\Omega \sin \psi = 0 \quad (4.4)$$

*i.e.*  $A_{11} = K_1 = K, B_{11} = 1, \psi_1 = \pi/2$

In each case it has been shown (Refs 1 and 2) that the values of  $\theta_\alpha, \theta_q\Omega$  are given approximately by

$$\theta_\alpha = \bar{v}^2 / (K^2 + \bar{v}^2) \quad (4.5)$$

$$\theta_q\Omega = K / (K^2 + \bar{v}^2) \quad (4.6)$$

Figs 4 and 5 show the variation of  $\theta_\alpha$  and  $\theta_q\Omega$  with the damping constant  $K$  for a frequency ratio of  $\bar{v} = 0.01$ . This value is of the same order of magnitude as that experienced by a helicopter when slightly disturbed from the equilibrium position. It can be seen that  $\theta_\alpha$  decreases rapidly with increasing  $K$  and that  $\theta_q\Omega$  reaches a maximum and then decays.

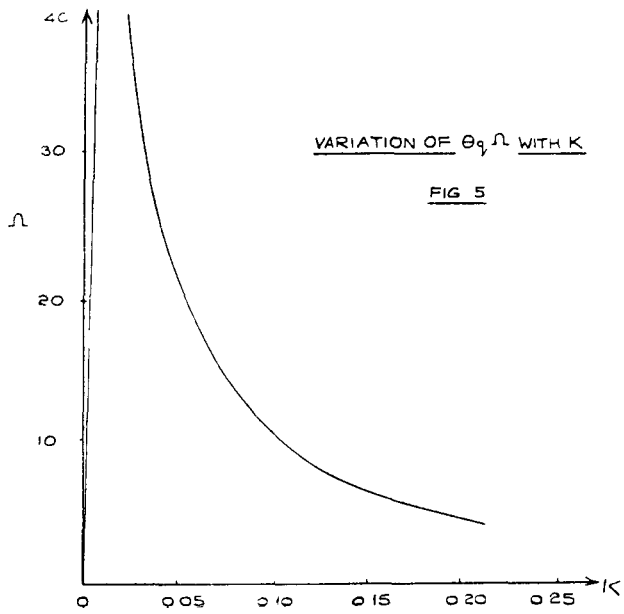
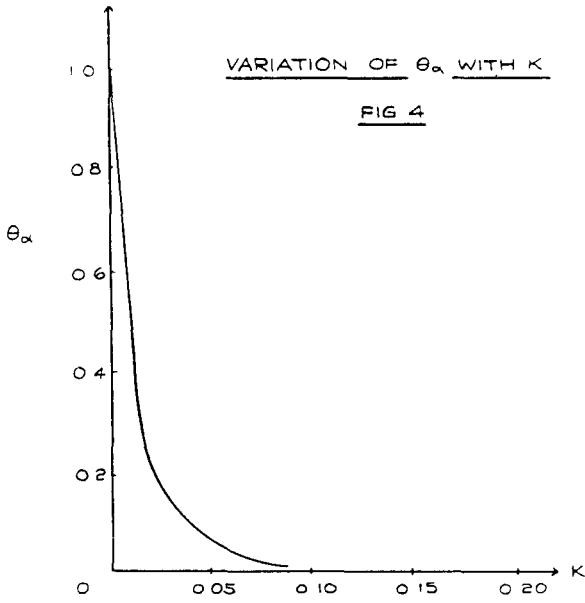
From the stability charts of Ref 1 we see that  $A_1/A_3$  and hence  $\theta_\alpha$  must be greater than zero. Therefore since the decrease of  $\theta_\alpha$  with  $K$  is fast we must have a small value of  $K$ . This leads to a large  $\theta_q\Omega$ . The minimum practical value for  $K$  is of the order 0.03.

In the first of the two cases the excitement due to the gyroscopic forces associated with the mass of the bar has a frequency which is practically equal to its natural frequency. Thus the motion must contain damping.

In the second case the damping is provided by the air forces. The specific damping of a typical rotor blade considering only air forces is approximately  $K = \gamma_0/16$ . The damping is still too large even if we use very heavy blades, *i.e.*, for small values of  $\gamma_0$ . In order therefore to obtain the necessary small amount of damping the servo-blade, located at the outer part of the radius, is relatively short, *i.e.*, of small aspect ratio.

For the value of  $K = 0.03$  we see that  $\theta_\alpha = 0.10$  and  $\theta_q\Omega = 30$ . Unfortunately these values only serve to give neutral stability. This can be seen from the stability charts of Ref 1 where for a given value of  $A_1/A_3$  and  $A_0/A_3$  an increase in  $A_2/A_3$  leads to an increase in the time for a disturbance oscillation to decay to half its initial amplitude.

The large value of  $\theta_q\Omega$  has also a detrimental effect upon the control sensitivity. In his lecture to the Helicopter Association of Great Britain in 1948 (Ref 3) Sisingh gives a plot of control effectiveness against  $\theta_\alpha$  for various values of  $\theta_q\Omega$ . It is shown that control displacements in phase with the rate of change of attitude  $\theta_q$  play the major part in determining the effectiveness of the pilot's controls. An increase in  $\theta_q$  causes a decrease in the effectiveness. Although the definition of control sensitivity used, *viz.*, the ratio of the amplitude of the forced oscillation of the helicopter to





the amplitude of the manual control displacement when the pilot applies a manual periodic control of period 4 secs, is not a completely satisfactory criterion for judging the response of an automatically stabilized helicopter, it does show in a simple way that loss in control sensitivity is mainly caused by the component  $\theta_q$  becoming too large

The limitations of the system in current use can be summarized thus

- i  $\theta_a$  is too small
- ii  $\theta_q \Omega$  is too large

For these systems the ratio  $\frac{\theta_q \Omega}{\theta_a}$  is of the order of 300, good stability can be obtained if this ratio is about 30

These limitations are caused through the minimum practical value of  $K$  being too great, so let us consider the properties of a system with an additional degree of freedom

### 5 Second Order Systems ( $n=2$ )

The equations of motion for a second order system are

$$\begin{aligned} \delta_1 + 2\Omega A_{11}\delta_1 + 2\Omega A_{12}\delta_2 + \Omega^2 B_{11}\delta_1 + \Omega^2 B_{12}\delta_2 \\ + 2\Omega q \sin(\psi + \psi_1) - q \cos(\psi + \psi_1) - 2qK_1\Omega \cos(\psi_1 + \psi) = 0 \end{aligned}$$

and

$$\begin{aligned} \delta_2 + 2\Omega A_{21}\delta_1 + 2\Omega A_{22}\delta_2 + \Omega^2 B_{21}\delta_1 + \Omega^2 B_{22}\delta_2 \\ + 2\Omega q \sin(\psi + \psi_2) - q \cos(\psi + \psi_2) - 2qK_2\Omega \cos(\psi + \psi_2) = 0 \end{aligned}$$

and the variations of the blade pitch angle are given by

$$\Delta\theta = G(\delta_1 + \bar{n}\delta_2) - G(\theta_{1s} + \bar{n}\theta_{2s}) \sin\psi \quad (5\ 2)$$

where  $\bar{n} = n_2/n_1$

From these equations we obtain

$$\theta_a'G = \frac{L\bar{v}^2}{M + N\bar{v}^2} + 0(\bar{v}^4) \quad (5\ 3)$$

and

$$\theta_q\Omega/G = \frac{P + Q\bar{v}^2}{M + N\bar{v}^2} + 0(\bar{v}^4) \quad (5\ 4)$$

where the constants  $P$ ,  $M$ ,  $Q$  and  $N$  are functions of the parameters of the system defined by (5 1). They are to be chosen such that terms in powers of  $\bar{v}$  greater than the third can be neglected. The detailed account of the derivation of equations 5 3 and 4 is given in Appendix II

From these equations it is seen that in order to reduce the magnitude of  $\theta_q\Omega$  in comparison with  $\theta_a$  we have to make  $P$  small since it is generally impossible to make  $L$  sufficiently large. We shall put  $P = 0$  and consider

the position where we have only mechanical damping at the hinge with no interconnecting moments but with varying azimuth angles. Afterwards we shall introduce springs and aerodynamic damping and consider their effect on the results.

Equations (5.1) become for a system with only mechanical damping

$$\delta_1 + 2\Omega A_{11}\delta_1 + \Omega^2\delta_1 + 2\Omega q \sin(\psi + \psi_1) - q \cos(\psi + \psi_1) = 0$$

and

$$\delta_2 + 2\Omega A_{22}\delta_2 + \Omega^2\delta_2 + 2\Omega q \sin(\psi + \psi_2) - q \cos(\psi + \psi_2) = 0 \quad (5.5)$$

In Appendix III it is shown that

$$\theta_a/G\bar{v}^2 = \frac{(A_{22} - A_{11}) \sin \psi_1 \sin \psi_2 + \frac{1}{2}A_{11}A_{22} \sin(\psi_1 - \psi_2)}{A_{11}^2 A_{22} \sin \psi_2} \quad (5.6)$$

and

$$\theta_q\Omega/G\bar{v}^2 = \frac{A_{22} \sin \psi_2 \cos \psi_1 - A_{11} \sin \psi_1 \cos \psi_2 + \left(\frac{A_{11}}{A_{22}} - \frac{A_{22}}{A_{11}}\right) \sin \psi_1 \sin \psi_2}{A_{11}^2 A_{22} \sin \psi_2} \quad (5.7)$$

where we also neglect  $N\bar{v}^2$  when compared with  $M$  and the value of  $\bar{n}$  required to give  $P = 0$  is

$$\bar{n} = -\frac{A_{22} \sin \psi_1}{A_{11} \sin \psi_2} \quad (5.8)$$

The constant  $G$  in these equations is a gearing ratio and increases the value of  $\theta_a$  and  $\theta_q\Omega$  whilst leaving the ratio  $\theta_q\Omega/\theta_a$  constant. It is considered to be positive.

The neglected terms in the series expansions used to determine  $\theta_a$  and  $\theta_q\Omega$  have no significant effect. This is because of the low value of  $\bar{v}$  which is approximately 0.01 for a typical helicopter.

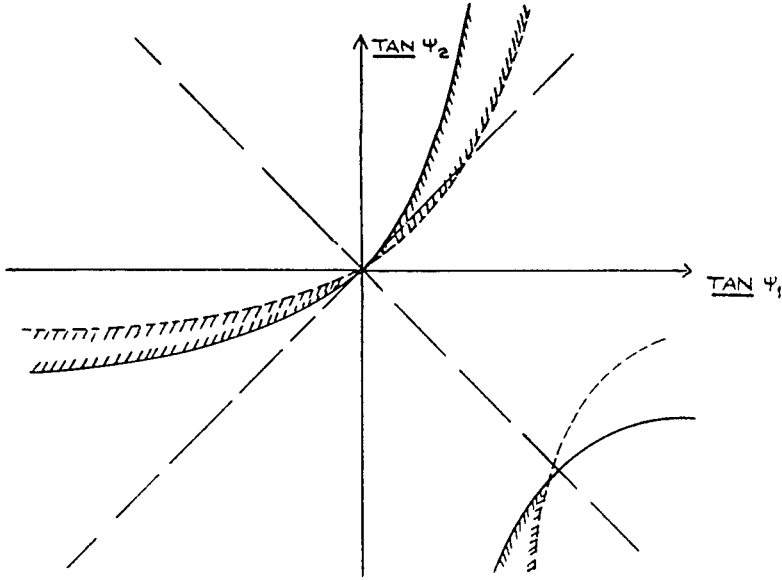
From the form of equations (5.6) and (5.7) we see that  $\theta_a$  and  $\theta_q\Omega$  are not necessarily positive. It can be seen from the stability charts of Ref. 1 that for the values of the coefficients of the stability equation (2.1) used, we must have both  $\theta_a$  and  $\theta_q\Omega$  positive in order to improve the dynamic stability.

Let us therefore consider the boundaries where  $\theta_a$  and  $\theta_q\Omega$  are zero in both the  $(\psi_1 - \psi_2)$  plane and the  $(A_{11} - A_{22})$  plane. The areas in these planes where  $\theta_a$  and  $\theta_q\Omega$  are both greater than zero will be called 'available regions'.

Figs. 6 and 7 show the boundaries and available regions for the range  $-\pi/2 < \psi_1, \psi_2 < \pi/2$ . We have plotted for convenience  $\tan \psi_1$  and  $\tan \psi_2$ . It can be seen that for  $A_{22} > A_{11}$  there are regions for  $\psi_1, \psi_2 < 0$ ,  $\psi_1, \psi_2 > 0$  and  $\psi_1 > 0, \psi_2 < 0$ . For  $A_{22} < A_{11}$  the only region occurs when  $\psi_1 < 0$  and  $\psi_2 > 0$ .

KEY	
//////	AVAILABLE REGION
—	$\Theta_\alpha = 0$
---	$\Theta_\psi = 0$
- - -	$\psi_1 - \psi_2 = 0$
- - -	$\psi_1 + \psi_2 = 0$

FIG 6



AVAILABLE REGIONS FOR  $-\frac{\pi}{2} < \psi_1, \psi_2 < \frac{\pi}{2}$  AND  $n > 1$

In Figs 8 and 9 we have moved the origin of the azimuth angle through a right angle by substituting

$$\psi_1 = \phi_1 + \pi/2 \quad (5.9)$$

$$\text{and } \psi_2 = \phi_2 + \pi/2 \quad (5.10)$$

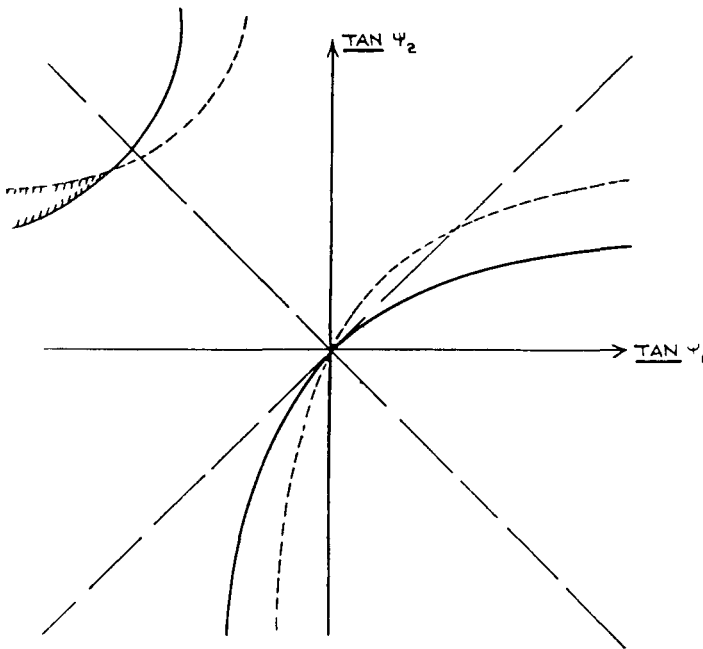
From these diagrams we see that for  $A_{22} > A_{11}$ , regions occur for  $\psi_1 < \pi/2$ ,  $\psi_2 < \pi/2$  and  $\psi_1 < \pi/2$ ,  $\psi_2 > \pi/2$ . For  $A_{22} < A_{11}$  we have regions for  $\psi_1 > \pi/2$ ,  $\psi_2 > \pi/2$  and  $\psi_1 > \pi/2$ ,  $\psi_2 < \pi/2$ .

The diagrams are only sketches of the shape of the boundaries which are expected. The position and gradients are functions of both  $A_{11}$  and  $A_{22}$ . The theory behind the curves is given in Appendix IV. If we now plot the boundaries in the damping plane for given values of  $\psi_1$  and  $\psi_2$  we

have only to consider positive values of  $A_{11}$  and  $A_{22}$ . Figs (10—16) show the available regions when  $A_{22}/A_{11}$  is plotted against  $A_{11}$ . Fig 10 is for the case where  $-\pi/2 < \psi_1 < \psi_2 < 0$ , Fig 11 shows the regions when  $-\pi/2 < \psi_1 < 0 < \psi_2 < \pi/2$  and  $\psi_1 + \psi_2 < 0$ . The region for both  $-\pi/2 < \psi_2 < 0 < \psi_1 < \pi/2$ ,  $\psi_1 + \psi_2 < 0$  and  $0 < \psi_1 < \pi/2 < \psi_2 < \pi$ ,  $\psi_1 + \psi_2 < \pi$  is shown in Fig 12. In Fig 13 we show the region for  $0 < \psi_2 < \psi_1 < \pi/2$  and in Fig 14 the region for  $0 < \psi_1 < \psi_2 < \pi/2$ . When  $0 < \psi_2 < \pi/2 < \psi_1 < \pi$  and  $\psi_1 + \psi_2 > \pi$  the available region is given in Fig 15. Finally in this section, Fig 16 shows the region for  $\pi/2 < \psi_2 < \psi_1 < \pi$ . These results follow from the data in Appendix V and it can be shown that there are no available regions for the following

KEY	
////	AVAILABLE REGION
—	$\Theta_\alpha = 0$
—	$\Theta_\psi = 0$
- - -	$\psi_1 - \psi_2 = 0$
- - -	$\psi_1 + \psi_2 = 0$

FIG 7



AVAILABLE REGIONS FOR  $-\pi/2 < \psi_1, \psi_2 < \pi/2$  AND  $m < 1$

$$-\pi/2 < \psi_2 < \psi_1 < 0$$

$$-\pi/2 < \psi_1 < 0 < \psi_2 < \pi/2 \text{ with } \psi_1 + \psi_2 > 0$$

$$-\pi/2 < \psi_2 < 0 < \psi_1 < \pi/2 \text{ with } \psi_1 + \psi_2 > 0$$

$$0 < \psi_1 < \pi/2 < \psi_2 < \pi \text{ with } \psi_1 + \psi_2 > \pi$$

$$0 < \psi_2 < \pi/2 < \psi_1 < \pi \text{ with } \psi_1 + \psi_2 < \pi$$

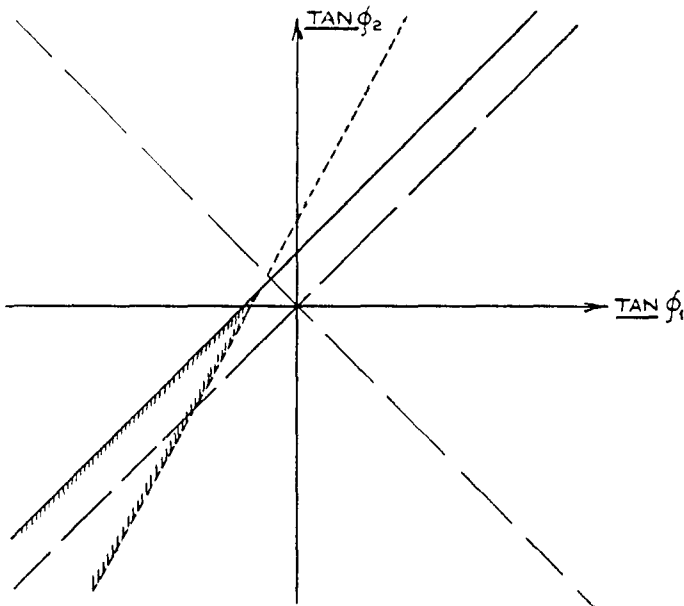
and

$$\pi/2 < \psi_1 < \psi_2 < \pi$$

On comparing Figs 6—9 with Figs 10—16 we see that they correspond very closely. The regions for  $A_{22} > A_{11}$  given in Fig 6 correspond to

KEY	
/ /	AVAILABLE REGION
—	$\theta_x = 0$
—	$\theta_y = 0$
— —	$\phi_1 - \phi_2 = 0$
— —	$\phi_1 + \phi_2 = 0$
$\phi$	$= \psi_1 - \frac{\pi}{2}$
$\phi_2$	$= \psi_2 - \frac{\pi}{2}$

FIG 8



AVAILABLE REGIONS FOR  $0 < \psi_1, \psi_2 < \pi$  AND  $m > 1$

Figs 10, 12, 13 and 14 The region for  $A_{11} > A_{22}$  given in Fig 7 corresponds to Fig 11 The regions of Fig 8 correspond to Figs 12, 13 and 14 whilst those of Fig 9 are equivalent to Figs 15 and 16 It can be seen also that the regions given above where no available regions exist correspond to similar regions in Figs 6—9

### 6 Three Special Cases

Let us now consider some special cases Firstly let  $A_{11} = 0.30$  and  $A_{22} = 0.35$  Fig 17 shows the available regions near the origin and it can be seen that it is of the form predicted by Fig 6 The assumption that the second term in the denominator of the expressions (5.3) and (5.4) for  $\theta_a/G$  and  $\theta_q\Omega/G$  is justified for, with the given values of  $A_{11}$  and  $A_{22}$  and a

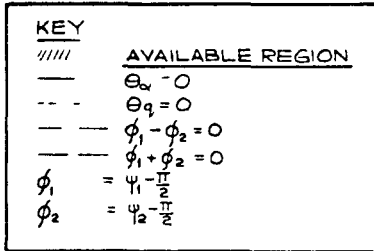
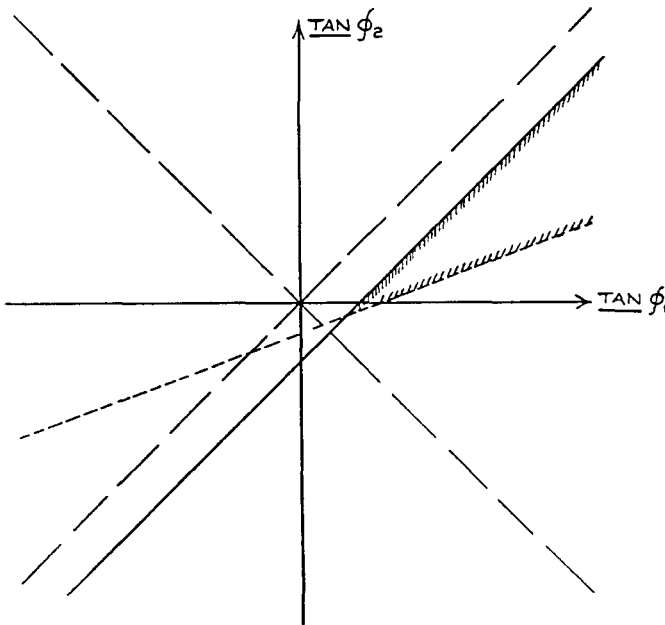


FIG 9



AVAILABLE REGION FOR  $0 < \psi_1, \psi_2 < \pi$  AND  $m < 1$

frequency ratio of about 0.01, it can easily be proved that  $N\bar{v}^2/M$  is of the order of 0.005. Effects of this order of magnitude can be neglected.

For the second example we have chosen  $\psi_1 = -\pi/3$  and  $\psi_2 = -\pi/6$ . In Fig. 18  $m = A_{22}/A_{11}$  is plotted against  $A_{11}$  and the available region corresponds to the expected area given by Fig. 10.

Thirdly, we have chosen  $\psi_1 = \pi/6$  and  $\psi_2 = \pi/3$ . The available region Fig. 19 agrees with the predicted region Fig. 14. If we now choose  $A_{22} = 2A_{11}$  and plot  $\theta_\alpha/G\bar{v}^2$ ,  $\theta_q\Omega/G\bar{v}^2$  and  $\theta_q\Omega/\theta_\alpha$  against  $A_{11}$  we obtain Fig. 20. From this figure it can be seen that small values of the ratio  $\theta_q\Omega/\theta_\alpha$  are possible and hence good stability characteristics can be obtained. We now introduce springs and aerodynamic damping into the system and investigate their effect. Let us first consider the spring effect.

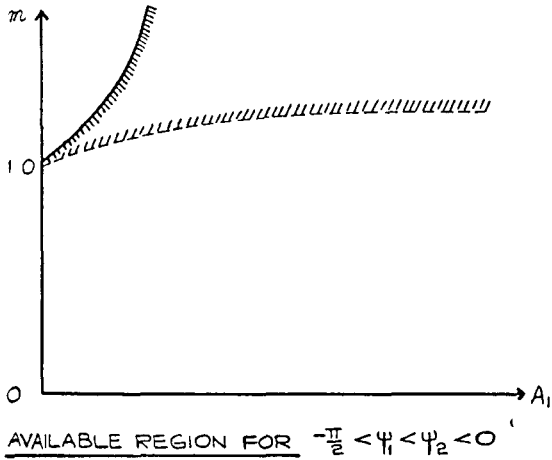


FIG. 10

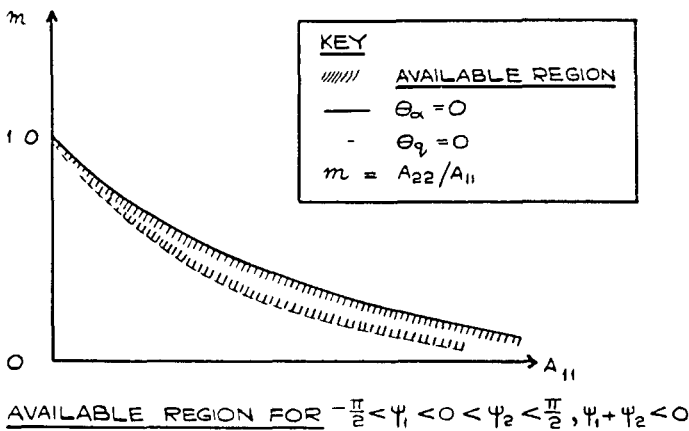


FIG. 11

## 7 Effect of Spring Restraint

The equation of motion can now be written in the form

$$\delta_i + 2\Omega A_{ii}\delta_i + \Omega^2(1 + \Delta B_{ii})\delta_i + 2\Omega q \sin(\psi + \psi_i) - q \cos(\psi + \psi_i) = 0 \quad (i = 1, 2) \quad (7.1)$$

The original  $B_{ii}$  of equations (5.1) has been replaced by  $(1 + \Delta B_{ii})$  for convenience in the algebra and because  $\Delta B_{ii} = 0$  reduces (7.1) to the equations (5.5) where the spring effect is absent. By conducting a similar analysis as performed in Appendix III it can be shown that both  $\theta_\alpha$  and  $\theta_q\Omega$  are of the form

$$\frac{(\alpha_1 \tan \psi_1 \tan \psi_2 + \alpha_2 \tan \psi_1 + \alpha_3 \tan \psi_2 + \alpha_4) \cos \psi_1}{\beta_1 + \beta_2 \tan \psi_2} \quad (7.2)$$

where  $\alpha_j$  and  $\beta_k$  are functions of both  $A_{ii}$  and  $\Delta B_{ii}$  ( $j = 1, 2, 3, 4, k = 1, 2$ ).

Thus the major changes in the picture for the available regions in the  $(\psi_1, \psi_2)$  plane are that the curve  $\theta_\alpha = \theta_q\Omega = 0$  suffer changes in their asymptotes. Let us consider a numerical example. Taking

$$A_{11} = 0.30, A_{22} = 0.35, \Delta B_{11} = 0, \Delta B_{22} = 0.10$$

and considering that part of the  $\psi$  plane where  $0 < \psi_1, \psi_2 < \pi/2$  we obtain Fig. 21. From this figure we see that larger values of  $\psi_1$  may now be used and that the previous values when the spring effect is absent are no longer 'available'.

In Fig. 22 we have plotted  $\theta_\alpha/G\bar{v}^2$ ,  $\theta_q\Omega/G\bar{v}^2$  and  $\theta_q\Omega/\theta_\alpha$  against  $\tan \psi_1$  for  $\psi_2 = \pi/4$ . The figure shows that in the new available region the desired value of  $\theta_q\Omega/\theta_\alpha$  necessary for good dynamic stability can be achieved. Thus we see that the introduction of the spring effect does not harm the system's ability to produce the desired ratio but only alters the position of the available areas.

## 8 Effect of Aerodynamic Damping

On the introduction of damping due to air forces and neglecting spring effects the equation of motion becomes

$$\delta_i + 2\Omega(A_{ii} + K_i)\delta_i + \Omega^2\delta_i + 2\Omega q \sin(\psi + \psi_i) - q \cos(\psi + \psi_i) - 2qK_i\Omega \cos(\psi + \psi_i) = 0 \quad (i = 1, 2) \quad (8.1)$$

The aerodynamic effect is given by the  $K_i$  terms and it can be seen that when  $K_1 = K_2 = 0$ , equation (8.1) reduces to equations (5.5) where only mechanical damping at the hinge is considered.

The effect on the boundaries  $\theta_\alpha = \theta_q\Omega = 0$  is the same as in the case of the introduction of springs into the system.

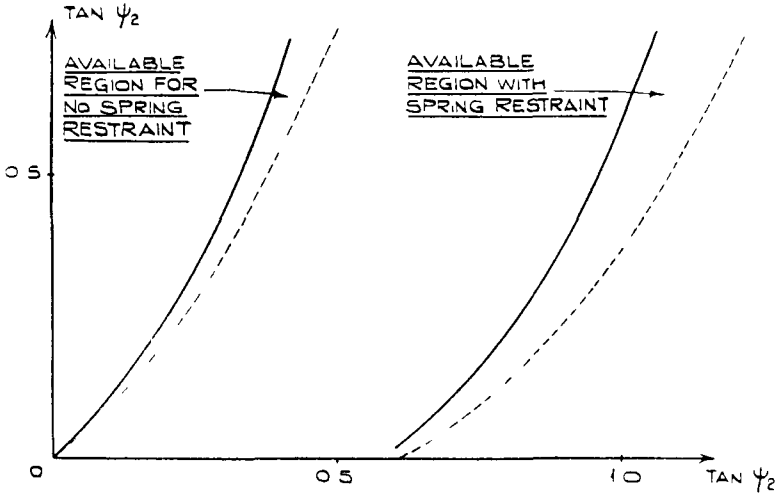
Let us consider the case where

$$A_{11} = 0.30, A_{22} = 0, K_1 = 0, K_2 = 0.35$$

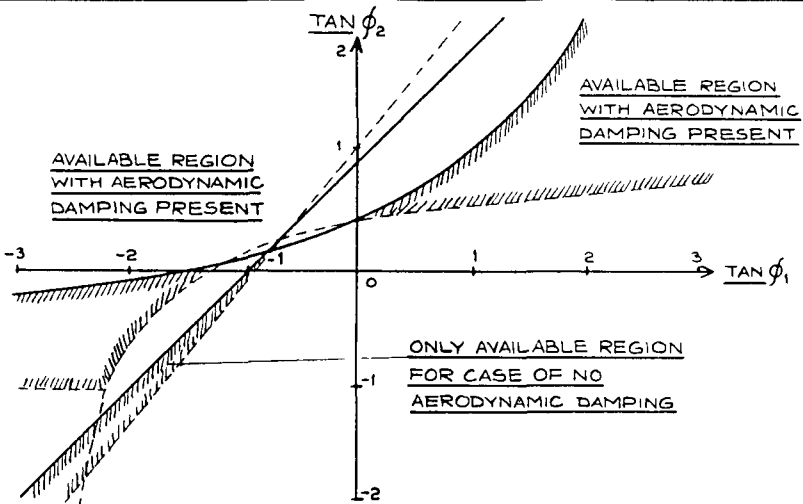


KEY —  $\theta_\alpha = 0$   
 - - -  $\theta_q = 0$

FIG 21



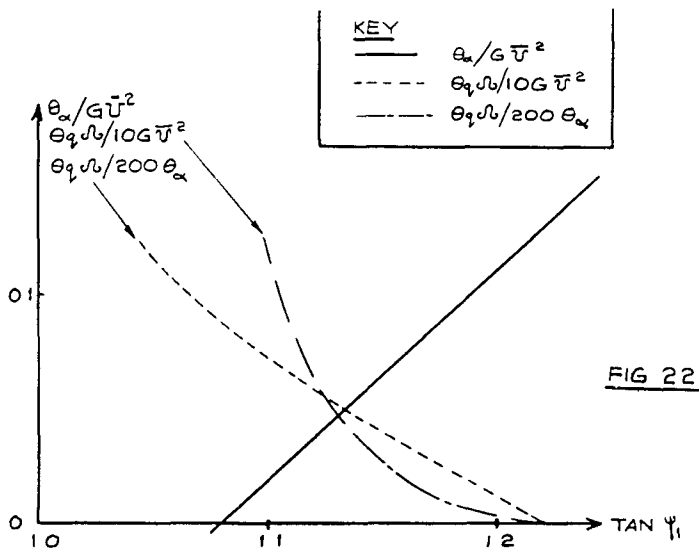
THE EFFECT OF SPRING RESTRAINT ON AVAILABLE REGIONS FOR  $0 < \psi_1, \psi_2 < \pi/2$   
 $A_{11} = 0.30, A_{22} = 0.35, \Delta B_{11} = 0, \Delta B_{22} = 0.10$



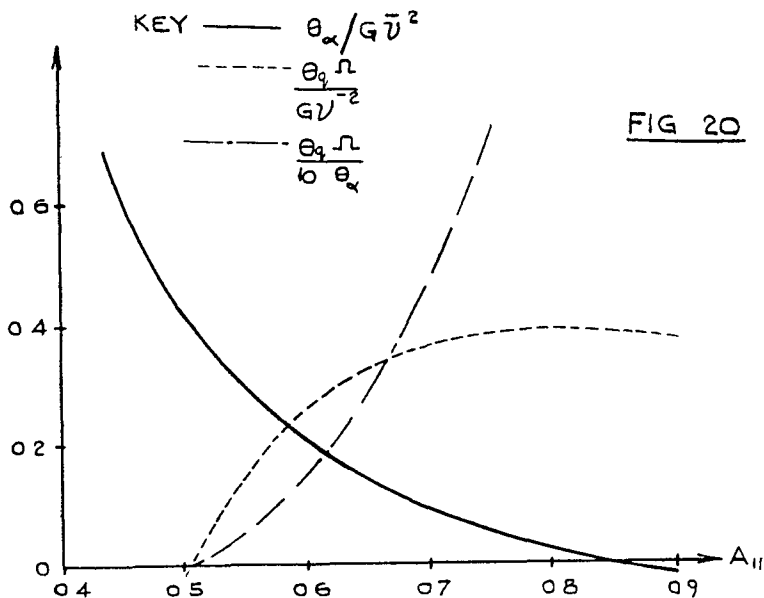
THE EFFECT OF AERODYNAMIC DAMPING ON AVAILABLE REGIONS

FOR  $0 < \psi_1, \psi_2 < \pi, A_{11} = 0.30, A_{22} = 0, K_1 = 0, K_2 = 0.35$

FIG 23



VARIATIONS OF  $\theta_\alpha / G \bar{U}^2$ ,  $\theta_q \Omega / G \bar{U}^2$  AND  $\theta_q \Omega / \theta_\alpha$  WITH  $\tan \psi_1$   
 FOR  $A_{11} = 0.30$ ,  $A_{22} = 0.35$ ,  $\Delta B_{11} = 0$ ,  $\Delta B_{22} = 0.10$ ,  $\tan \psi_2 = 1.0$



VARIATIONS OF  $\theta_\alpha / G \bar{U}^2$ ,  $\frac{\theta_q \Omega}{G \bar{U}^2}$ ,  $\frac{\theta_q \Omega}{10 \theta_\alpha}$  WITH  $A_{11}$

FOR  $\psi_1 = \pi/6$ ,  $\psi_2 = \pi/3$ ,  $m = 2$

This corresponds to a system where one rod is mechanically damped at the hinge and the other is damped aerodynamically. Fig 23 shows the changes produced in the  $\psi$  plane where  $0 < \psi_1, \psi_2 < \pi$ . It can be seen that the available region before the introduction of aerodynamic damping has disappeared and has been replaced by two larger regions. The reason why the new curves fail to degenerate into straight lines as in the case of pure mechanical damping, is because of the introduction of the  $\alpha_4$  term in equation (7.2). In Fig 24 we have plotted  $\theta_\alpha/G\bar{v}^2$ ,  $\theta_q\Omega/G\bar{v}^2$  and  $\theta_q\Omega/\theta_\alpha$  against  $\tan \phi_1$  for  $\phi_2 = \frac{7}{4}\pi$ , i.e.,  $\psi_2 = \frac{3}{4}\pi$ . We see again that the required value of  $\theta_q\Omega/\theta_\alpha$  for good stability can be obtained.

## 9 Conclusions

1 Theoretically a helicopter can be fitted with an automatic control device utilizing the rotor shaft as the origin of gyroscopic couples to give any required stability characteristic with controls fixed. A second order system will give the small values of the ratio  $\theta_q\Omega/\theta_\alpha$  required for good stability.

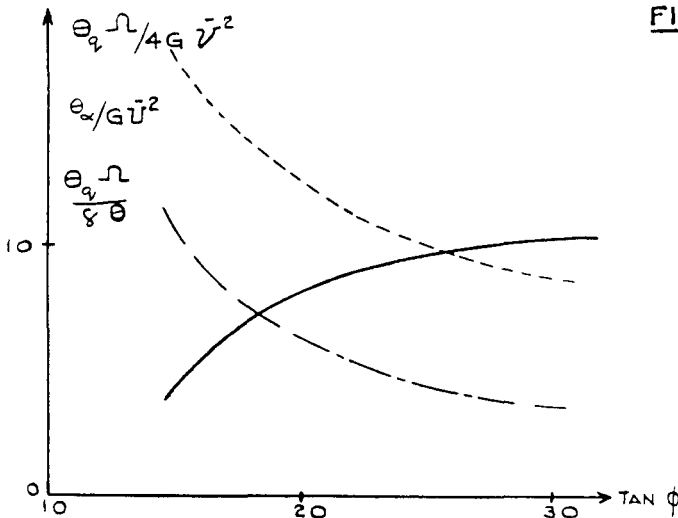
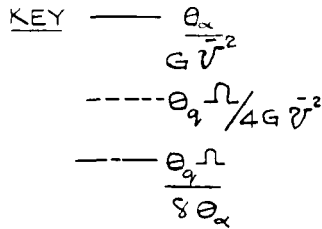


FIG 24

VARIATIONS OF  $\frac{\theta_\alpha}{G\bar{v}^2}$ ,  $\frac{\theta_q\Omega}{4G\bar{v}^2}$ ,  $\frac{\theta_q\Omega}{\theta_\alpha}$

FOR  $A_{11} = 0.30$ ,  $A_{22} = 0$ ,  $K_1 = 0$ ,  $K_2 = 0.35$ ,  $\tan \phi_2 = 1.0$

2 The position of the bars must be in 'available' regions The location of these regions can be changed by the introduction of aerodynamic damping and elastic restraint

3 The results are sufficient to indicate how mechanical apparatus should be designed for practical application of the principle

REFERENCES

- 1 'Investigation on automatic stabilization of the helicopter,' by G J Sissingh (R A E Rep Aero 2277)
- 2 'The Frequency Response of the Ordinary Blade, the Hiller Servo Blade and the Young-Bell Stabilizer,' by G J Sissingh (R and M 2860)
- 3 'Automatic Stabilization of Helicopters,' by G J Sissingh (Journal of the Helicopter Ass of Gt Britain, 1948)

APPENDIX I

*The Generalized 'Simple' Stabilization System*

We shall consider the system to be composed of n rods hinged at the axis of rotation and connected to some form of mass If we neglect the moment of inertia of the blades about their axes, the forces in the blade setting device which oppose the motion  $\delta, \dot{\delta}, \ddot{\delta}$ , friction and feedback to the separate rods due to their connection with the main control bar, the moments about the hinge give for the i th rod,

$$M_{ai} + M_{mi} + \sum_{j=1}^n M_{ij} = 0 \tag{A 1}$$

and  $M_{ai}$  = Moment due to Aerodynamic forces

$M_{mi}$  = Moment due to Mass forces

$M_{ij}$  = Moment due to the connection of j th rod with i th rod

$M_{ii}$  = Moment due to the connection of i th rod to the shaft

It is to be noted that in general  $M_{ij} \neq M_{ji}$

These separate moments are given by

$$M_{ai} = 2\Omega^2 K_1 I_{si} [\delta_i / \Omega + C' \delta_i - (q/\Omega) \cos(\psi + \psi_i)] \tag{A 2}$$

where

$I_{si}$  = Moment of Inertia of i th mass about hinge

$K_1$  = Specific damping and

$C'$  = coupling between the incidence of the i th mass and its angular displacement

$$M_{mi} = I_{si} [\delta_i + \Omega^2 \delta_i + 2\Omega q \sin(\psi + \psi_i) - q \cos(\psi + \psi_i)] \tag{A 3}$$

$$M_{ii} = 2K'_1 \Omega I_{si} \delta_i + \epsilon_1 \delta_i I_{si} \Omega^2 \tag{A 4}$$

where

$K'_1$  = specific damping at hinge of 1 th rod  
 $\epsilon_1$  = specific elastic constant

$$M_{1j} = S_{1j}(l_1\delta_1 - l_j\delta_j) + D_{1j}(l_1\delta_1 - l_j\delta_j) \quad (A 5)$$

$$= \Omega^2 I_{s1} (S_{1j}(l_1\delta_1 - l_j\delta_j) + \frac{2}{\Omega} d_{1j}(l_1\delta_1 - l_j\delta_j)) \quad (A 6)$$

assuming  $l_1^2 + l_j^2 - 2l_1l_j \cos(\psi_1 - \psi_j) < (l_1\delta_1 - l_j\delta_j)^2$  (A 7)

where  $S_{1j}$  and  $d_{1j}$  are specific spring and damping constants. The condition A 7 is imposed in order that the moment due to the interlinking forces acts in the same plane as the other moments. Fig 3 illustrates the situation.

Substituting these expressions into (A 1) gives equations of the form

$$\begin{aligned} \delta_i + 2\Omega \sum_{j=1}^n A_{ij} \delta_j + \Omega^2 \sum_{j=1}^n B_{ij} \delta_j + 2\Omega q \sin(\psi + \psi_i) \\ - q \cos(\psi + \psi_i) - 2qK_1\Omega \cos(\psi + \psi_i) = 0 \end{aligned} \quad (A 8)$$

(i = 1, 2, ..., n)

## APPENDIX II

The equations of motion for a Second Order System with no inter-connecting moments are

$$\begin{aligned} \delta_1 + 2\Omega\delta_1 A_{11} + \Omega^2 (1 + \Delta B_{11})\delta_1 + 2\Omega q \sin(\psi + \psi_1) - q \cos(\psi + \psi_1) \\ - 2K_1\Omega q \cos(\psi + \psi_1) = 0 \end{aligned} \quad (A 9)$$

$$\begin{aligned} \delta_2 + 2\Omega\delta_2 A_{22} + \Omega^2 (1 + \Delta B_{22})\delta_2 + 2\Omega q \sin(\psi + \psi_2) - q \cos(\psi + \psi_2) \\ - 2K_2\Omega q \cos(\psi + \psi_2) = 0 \end{aligned} \quad (A 10)$$

Substituting  $\delta_i = \Theta_{is} \sin \psi + \Theta_{ic} \cos \psi$  (i = 1, 2) (A 11)

and

$$\begin{aligned} a &= a_0 e^{i\omega t} \\ \Theta_{is} &= \theta_{is} e^{i\omega t} \\ \Theta_{ic} &= \theta_{ic} e^{i\omega t} \end{aligned} \quad (A 12)$$

we obtain

$$\frac{-\theta_{is}}{a_0} = \frac{\bar{v}^2(Z'_0 X'_2 + Z'_1 X'_1) + i\bar{v}Z'_0 X'_1 + \bar{v}^3(Z'_0 X'_3 + X'_1 Z'_2 - X'_2 Z'_1)}{Z'_0{}^2 + \bar{v}^2(Z'_1{}^2 + 2Z'_0 Z'_2)} \quad (A 13)$$

neglecting the terms in higher powers of  $\bar{v}$ ,

$$\text{where } Z'_0 = \Delta B_{11}^2 + 4A_{11}^2 \quad (\text{A } 14)$$

$$Z'_2 = -2\Delta B_{11} - 4A_{11}^2 - 4 \quad (\text{A } 15)$$

$$Z'_1 = 4A_{11} \Delta B_{11} + 8A_{11} \quad (\text{A } 16)$$

$$X'_2 = (-4 - \Delta B_{11} - 4A_{11}K_1) \sin \psi_1 \\ + (-2A_{11} + 4K_1) \cos \psi_1 \quad (\text{A } 17)$$

$$X'_1 = (4A_{11} + 2\Delta B_{11}K_1) \sin \psi_1 + (-4A_{11}K_1 + 2\Delta B_{11}) \cos \psi_1 \quad (\text{A } 18)$$

$$\text{and } X'_3 = (-2A_{11} - 2K_1) \sin \psi_1 \quad (\text{A } 19)$$

Similarly we obtain  $\theta_{2s}/\alpha_0$

$$\text{Now } \theta_{1a} = -\text{Re} (\theta_{1s}/\alpha_0) \quad (\text{A } 20)$$

$$\theta_{1q}\Omega\bar{v} = -\text{Im} (\theta_{1s}/\alpha_0) \quad (\text{A } 21)$$

$$\text{and } \Delta\theta = G (\delta_1 + \bar{n}\delta_2) \quad (\text{A } 22)$$

$$\text{so that } \theta_a = G (\theta_{1a} + \bar{n}\theta_{2a}) \quad (\text{A } 23)$$

$$\theta_q\Omega = G (\theta_{1q} + \bar{n}\theta_{2q}) \Omega \quad (\text{A } 24)$$

$$\text{Hence we can write } \theta_a/G = L\bar{v}^2/(M + N\bar{v}^2) \quad (\text{A } 25)$$

$$\text{and } \theta_q\Omega/G = \frac{P + Q\bar{v}^2}{M + N\bar{v}^2} \quad (\text{A } 26)$$

$$\text{where } L = [Z_0''(Z'_0X'_2 + Z'_1X'_1) + \bar{n}Z_0''(Z''_0X''_2 + Z''_1X''_1)] \quad (\text{A } 27)$$

$$P = Z'_0Z''_0[X'_1Z''_0 + \bar{n}Z'_0X''_1] \quad (\text{A } 28)$$

$$Q = [Z_0''^2(Z'_0X'_3 + Z'_2X'_1 - Z'_1X'_2) + Z'_0X'_1(Z_1''^2 + 2Z_0''Z_2'')] \\ + \bar{n}Z_0''X''_1(Z_1''^2 + 2Z_0''Z_2'') + \bar{n}Z_0''(Z''_0X''_3 + X''_1Z''_2 - X''_2Z''_1)] \quad (\text{A } 29)$$

$$M = Z_0''Z_0''^2 \quad (\text{A } 30)$$

$$\text{and } N = Z_0''(Z_1''^2 + 2Z_0''Z_2'') + Z_0''^2(Z_2''^2 + 2Z_0''Z_2'') \quad (\text{A } 31)$$

The double primed terms are the equivalent expressions of A (14—19) when the second bar is considered

Since, from A 14,  $Z' \neq 0$  except when both  $\Delta B_{11}$  and  $A_{11}$  are zero, we must have, for  $P = 0$ ,

$$X'_1Z''_0 + \bar{n}Z'_0X''_1 = 0 \quad (\text{A } 32)$$

APPENDIX III

When we consider only damping at the hinge A 14—19 become

$$Z'_0 = 4A_{11}^2 \tag{A 33}$$

$$Z'_2 = -4(1 + A_{11}^2) \tag{A 34}$$

$$Z'_3 = 8A_{11} \tag{A 35}$$

$$X'_2 = -4 \sin \psi_1 - 2A_{11} \cos \psi_1 \tag{A 36}$$

$$X'_1 = 4A_{11} \sin \psi_1 \tag{A 37}$$

and  $X'_3 = -2A_{11} \sin \psi_1 \tag{A 38}$

and the condition that P should be zero is that

$$\bar{n} = -\frac{A_{22} \sin \psi_1}{A_{11} \sin \psi_2} \tag{A 39}$$

When these expressions are substituted into equations A 27—30,

$$\frac{\theta_a}{G\bar{v}^2} = \frac{(A_{22} - A_{11}) \sin \psi_1 \sin \psi_2 + \frac{1}{2}A_{11}A_{22} \sin(\psi_1 - \psi_2)}{A_{11}^2 A_{22} \sin \psi_2} \tag{A 40}$$

and

$$\frac{\theta_q \Omega}{G\bar{v}^2} = \frac{A_{22} \sin \psi_2 \cos \psi_1 - A_{11} \sin \psi_1 \cos \psi_2 + \left(\frac{A_{11}}{A_{22}} - \frac{A_{22}}{A_{11}}\right) \sin \psi_1 \sin \psi_2}{A_{11}^2 A_{22} \sin \psi_2} \tag{A 41}$$

It is assumed that the ratio  $N\bar{v}^2/M$  is small and can be neglected

APPENDIX IV

*The Regions of Possible Azimuth Angles  $\psi_1, \psi_2$*

Let us consider the regions in the  $\psi_1, \psi_2$  plane where both  $\theta_a$  and  $\theta_q \Omega$  are positive

Let us write

$$F = (m - 1) \tan \psi_1 \tan \psi_2 + \frac{1}{2} m A_{11} (\tan \psi_1 - \tan \psi_2) \tag{A 42}$$

and  $f = -\tan \psi_1 + m \tan \psi_2 + \left(\frac{1 - m^2}{mA_{11}}\right) \tan \psi_1 \tan \psi_2 \tag{A 43}$

where  $m = A_{22}/A_{11} \tag{A 44}$

Then  $\frac{\theta_a}{G} = \frac{F \bar{v}^2 \cos \psi_1 \cos \psi_2}{A_{11}A_{22} \sin \psi_2} \tag{A 45}$

and  $\frac{\theta_q \Omega}{G} = \frac{f \bar{v}^2 \cos \psi_1 \cos \psi_2}{A_{11}A_{22} \sin \psi_2} \tag{A 46}$

Range I 
$$-\frac{\pi}{2} < \psi_1, \psi_2 < \frac{\pi}{2}$$

Here 
$$\cos \psi_1 \cos \psi_2 > 0 \tag{A 47}$$

Let 
$$e = \frac{1}{2}mA_{11}/(m - 1) \text{ and } h = \frac{mA_{11}}{m^2 - 1} \tag{A 48}$$

The boundaries are given by

$$F = 0 \text{ i e } \tan \psi_2 = \frac{e \tan \psi_1}{e - \tan \psi_1} \tag{A 49}$$

and 
$$f = 0 \text{ i e } \tan \psi_2 = \frac{h \tan \psi_1}{mh - \tan \psi_1} \tag{A 50}$$

Considering  $G > 0$  and using (A 47) we can write

$$\theta_a = \gamma^2 F / \sin \psi_2 \tag{A 51}$$

and 
$$\theta_q = \gamma'^2 f / \sin \psi_2 \tag{A 52}$$

where  $\gamma$  and  $\gamma'$  are real quantities

The condition  $G > 0$  corresponds to  $\Delta\theta$  being in the same sense as  $\delta$   
 The available areas are found by considering the various  $\psi_1, \psi_2$

(a)  $m > 1 \text{ i e } A_{22} > A_{11}$  The results are shown in fig 6

(b)  $m < 1 \text{ i e } A_{11} > A_{22}$  The results are shown in fig 7

Range II

$$0 < \psi_1, \psi_2 < \pi$$

Let 
$$\psi_1 = \phi_1 + \pi/2 \tag{A 53}$$

and 
$$\psi_2 = \phi_2 + \pi/2 \tag{A 54}$$

Then 
$$\theta_a / G\bar{v}^2 = \frac{F' \cos \phi_1}{A_{11}A_{22}} \tag{A 55}$$

and 
$$\theta_q \Omega / G\bar{v}^2 = \frac{f' \cos \phi_1}{A_{11}A_{22}} \tag{A 56}$$

where 
$$F' = (m - 1) + \frac{1}{2}mA_{11} (\tan \phi_1 - \tan \phi_2) \tag{A 57}$$

$$f' = \tan \phi_2 - m \tan \phi_1 + \left(\frac{1}{m} - m\right) / A_{11} \tag{A 58}$$



and the boundaries are given by

$$F' = 0 \text{ i e } \tan \phi_2 = \phi_1 + 1/e \quad (\text{A } 59)$$

and  $f' = 0 \text{ i e } \tan \phi_2 = m \tan \phi_1 + 1/h \quad (\text{A } 60)$

The results are illustrated in Fig 8 and 9. The regions are found by inspection

---

#### APPENDIX V

By writing  $a = \sin \psi_1 \quad (\text{A } 61)$

$$b = \frac{1}{2} \sin (\psi_1 - \psi) / \sin \psi_2 \quad (\text{A } 62)$$

$$c = \sin \psi_1 \cos \psi_2 / \sin \psi_2 \quad (\text{A } 63)$$

$$d = \cos \psi_1 \quad (\text{A } 64)$$

$$H = -a + ma + \frac{1}{2} m \Lambda_{11} b \quad (\text{A } 65)$$

and  $H' = -c + md + \left(\frac{1}{m} - m\right) a / \Lambda_{11} \quad (\text{A } 66)$

we see that  $\frac{\theta_\alpha}{G} = \frac{Hv^2}{\Lambda_{11}\Lambda_{22}} \quad (\text{A } 67)$

and  $\frac{\theta_q \Omega}{G} = \frac{H'v^2}{\Lambda_{11}\Lambda_{22}} \quad (\text{A } 68)$

By considering the groups—

- |    |  |
|----|--|
| 1  | $-\pi/2 < \psi_1 < \psi_2 < 0$         |
| 2  | $-\pi/2 < \psi_2 < \psi_1 < 0$         |
| 3  | $-\pi/2 < \psi_1 < 0 < \psi_2 < \pi/2$ |
| 4  | $-\pi/2 < \psi_2 < 0 < \psi_1 < \pi/2$ |
| 5  | $0 < \psi_1 < \psi_2 < \pi/2$          |
| 6  | $0 < \psi_2 < \psi_1 < \pi/2$          |
| 7  | $0 < \psi_1 < \pi/2 < \psi_2 < \pi$    |
| 8  | $0 < \psi_2 < \pi/2 < \psi_1 < \pi$    |
| 9  | $\pi/2 < \psi_1 < \psi_2 < \pi$        |
| 10 | $\pi/2 < \psi_2 < \psi_1 < \pi$        |

we obtain Figs 10—16

## LIST OF SYMBOLS

$A_i$	Coefficients of frequency equation ( $i = 0, 1, 2, 3$ )
$\theta_a$	automatic control component in phase with attitude
$\theta_q$	automatic control component in phase with rate of change of attitude
$\alpha$	angle in pitch (in radians)
$q$	rate of change of attitude with respect to time (in rads/sec)
$\psi$	azimuth angle measured from rear position in direction of rotation (in radians)
$\Omega$	angular velocity of rotor (in rads/sec)
$\delta$	automatic control device response
$\delta_i$	angular displacement of $i$ th rod of the device (in radians)
$\Delta\theta$	change of pitch setting (in radians)
$\theta_0$	mean pitch setting of rotor blade (in radians)
$G$	gearing ratio
$\psi_i$	azimuth angle of $i$ th rod measured from blade I (in radians)
$A_{ij}$	specific damping coefficient in generalized equations
$B_{ij}$	specific spring constant in generalized equations
$K_i$	specific aerodynamic damping in generalized equations
$\theta_s$	cyclic pitch component (in radians)
$i'_c$	ditto
$\nu$	frequency of the oscillation (in $\text{sec}^{-1}$ )
$\bar{\nu}$	$= \nu/\Omega$ frequency ratio
$\bar{n}_i, i = 1$	linkage ratios
$\gamma_0$	inertia number

Other symbols are defined in the text as required